

# eurac research

## Modelling results on New Generation Solar Cooling systems

Chiara Dipasquale



# INTRODUCTION

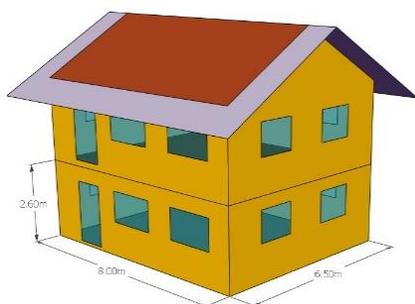
## 4 examples of new generation solar cooling systems:

- Building description and solar cooling plant layout;
- Working modes and characteristics of system components;
- Operational modes and system size variants, and results.

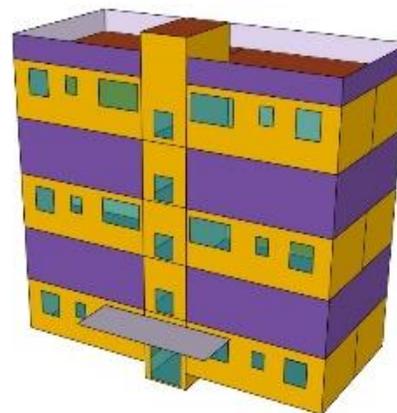
# CASE 1

## Building description

Reference Single Family House - SFH



Reference Small Multi Family House - sMFH

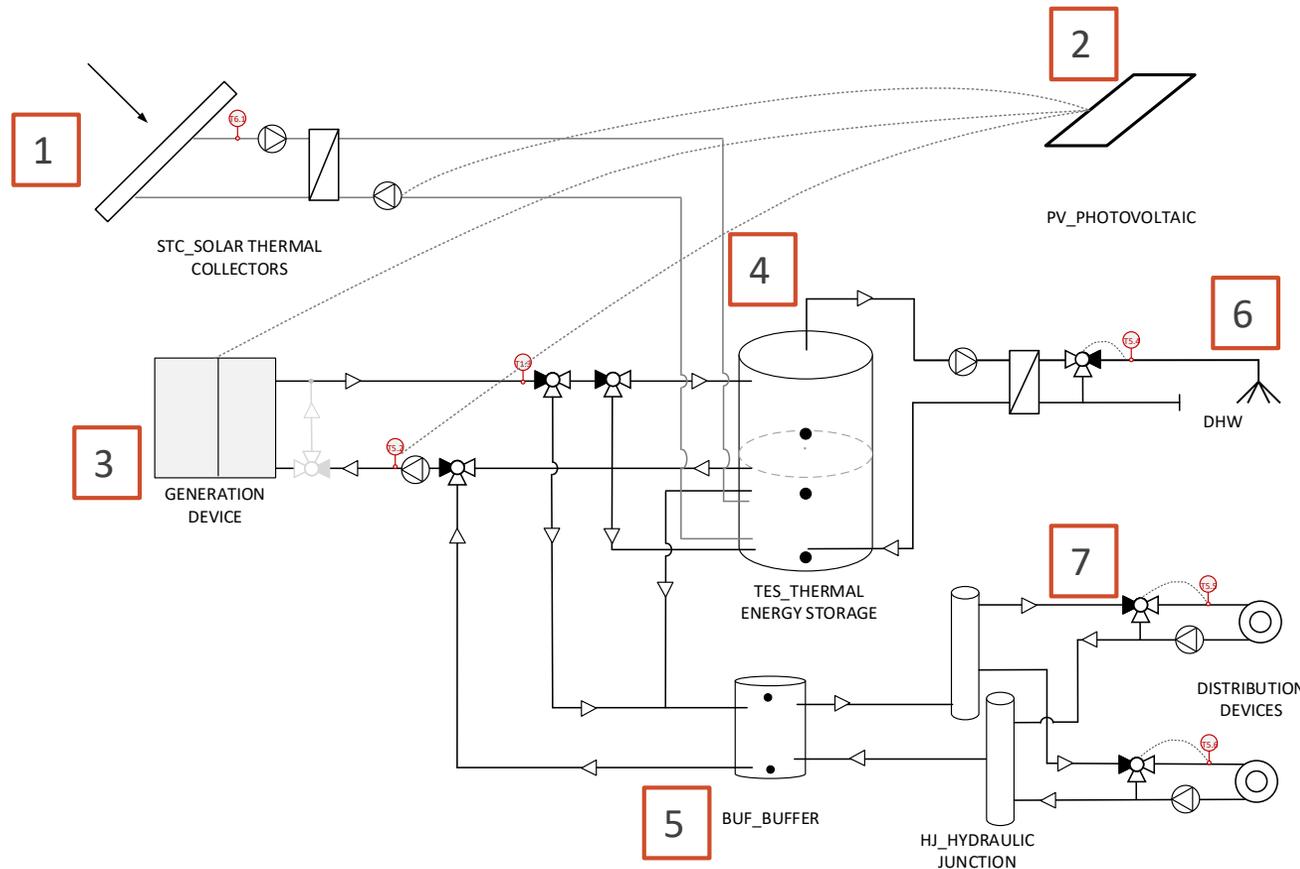


Number of floors	2
Living area per floor	50 m <sup>2</sup>
Yearly heating demand	45 kWh/(m <sup>2</sup> y)

Number of floors	5
Living area per dwelling	50 m <sup>2</sup>
Number dwelling per floor	2
Yearly heating demand	45 kWh/(m <sup>2</sup> y)

# CASE 1

## Solar cooling plant layout

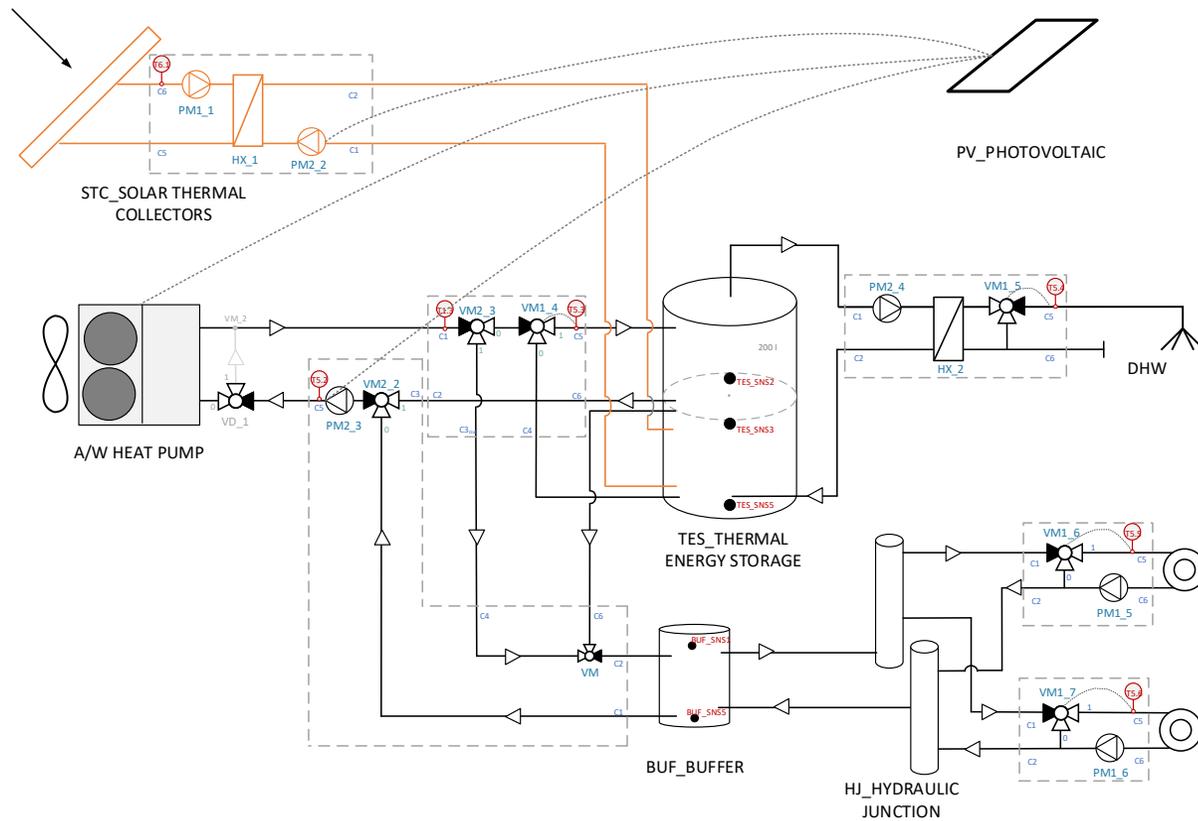


- Use of **solar thermal energy** for DHW production and space heating
- Use of **PV energy** for the HVAC system electricity consumption

1. Solar thermal collectors
2. PV panels
3. Air-to-water heat pump
4. Storage tank
5. Buffer
6. DHW distribution circuit
7. H&C Distribution circuit

# CASE 1

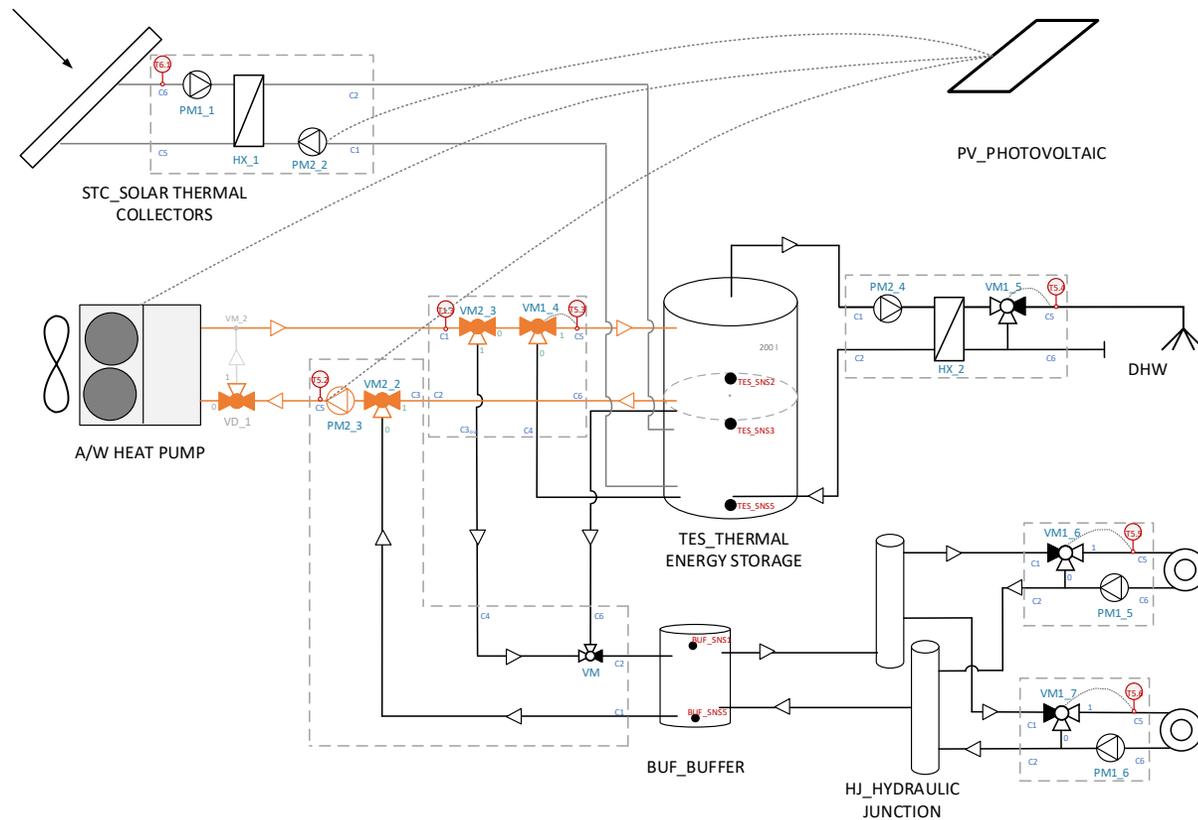
## Working conditions



TES charging by solar energy

# CASE 1

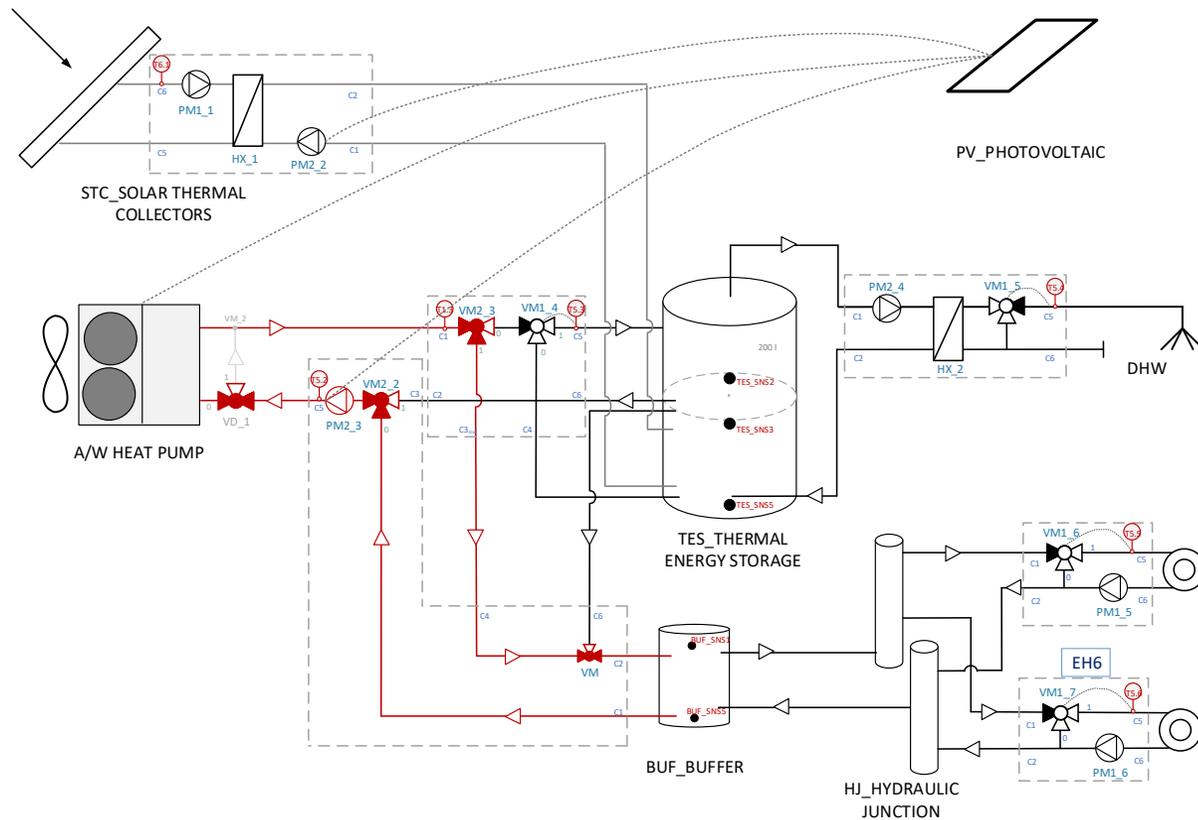
## Working conditions



TES charging by heat pump

# CASE 1

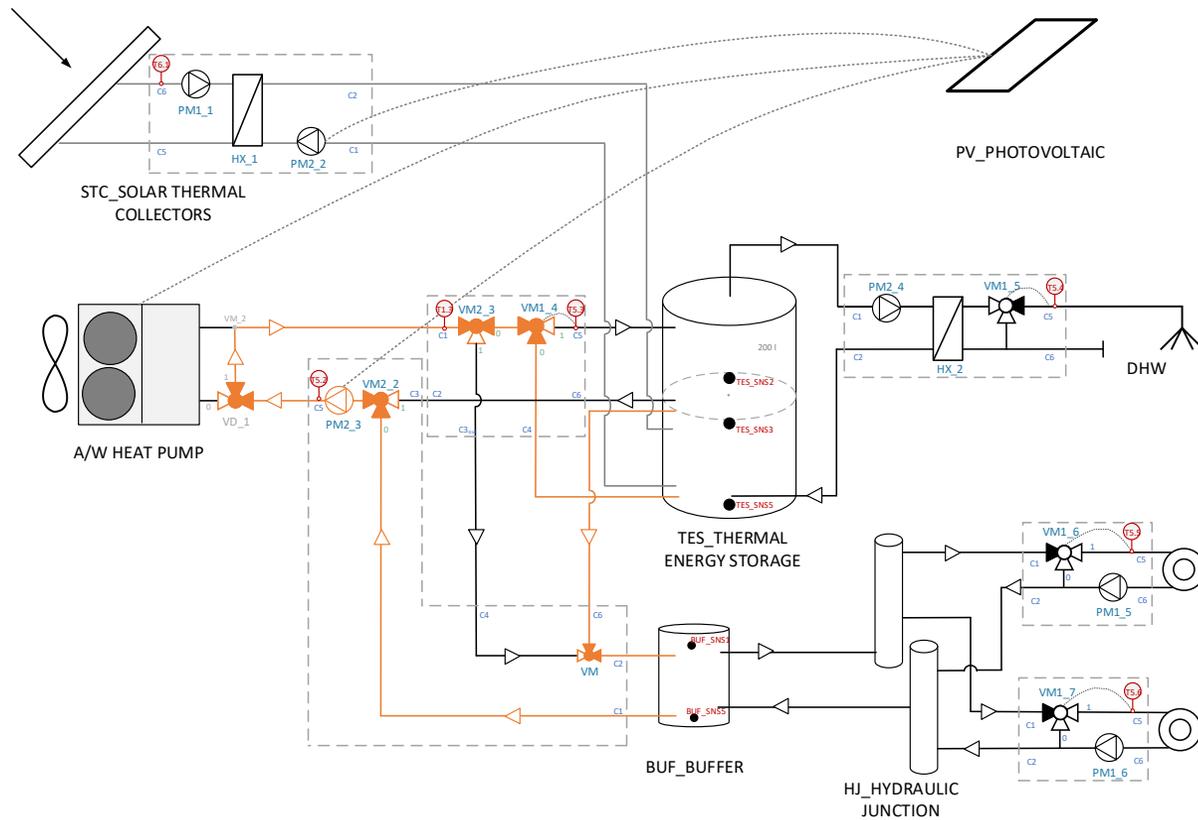
## Working conditions



Buffer charging by heat pump

# CASE 1

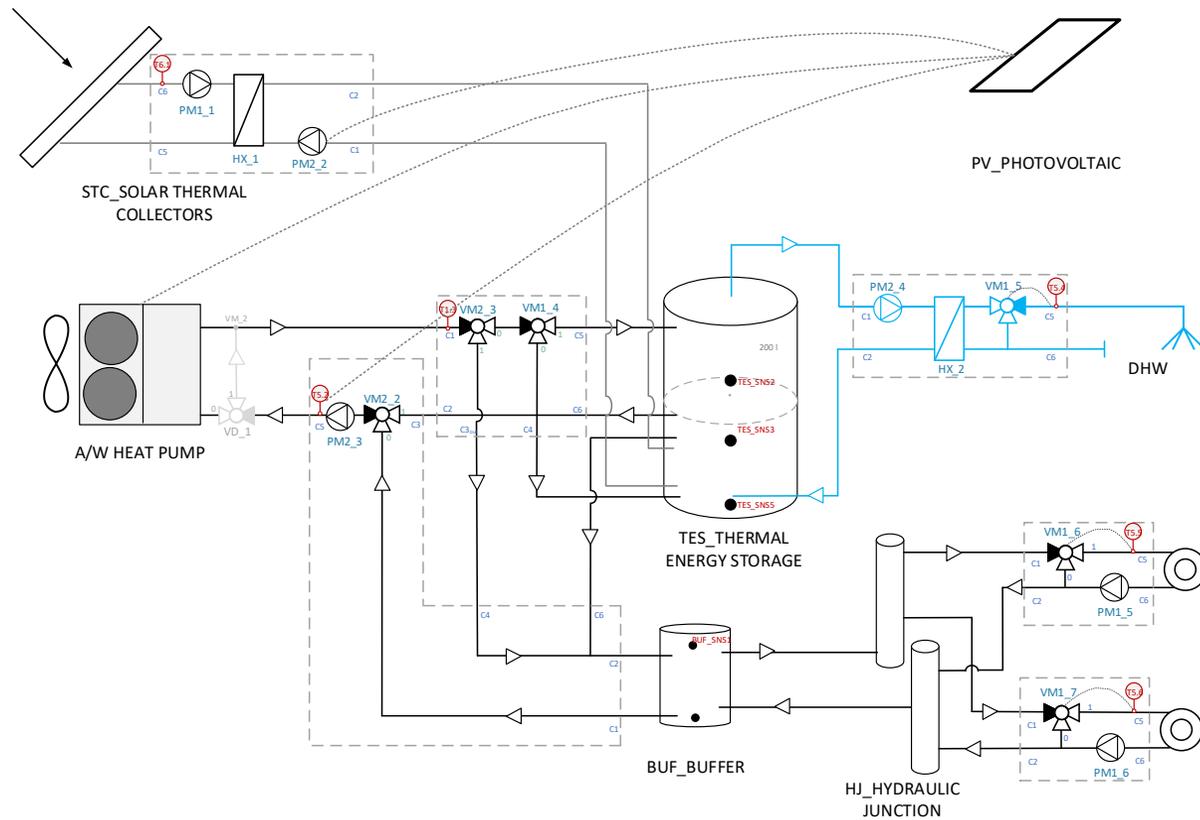
## Working conditions



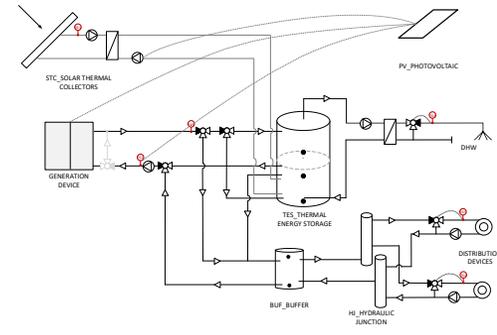
Buffer charging by solar energy

# CASE 1

## Working conditions



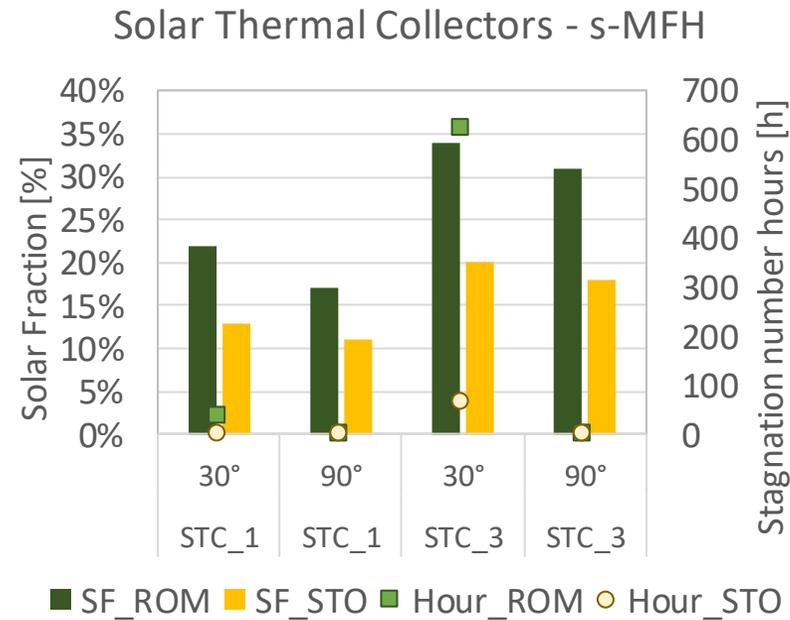
DHW distribution



# RESULTS – CASE 1

## ST and PV performance with varying field size and tilt angle

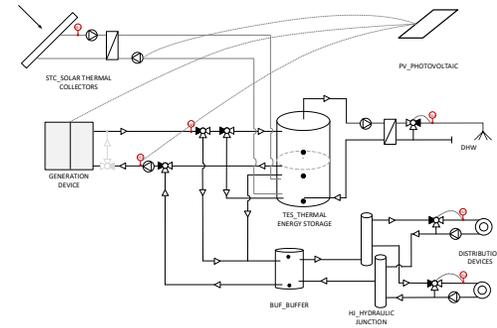
Solar Thermal		
	Unit	s-MFH
STC_1	m <sup>2</sup>	18.4
STC_2	m <sup>2</sup>	27.6
STC_3	m <sup>2</sup>	36.8



ROM – Rome

STO - Stockholm

Solar Fraction and stagnation hours referred to the total heating production (**space heating + DHW**)

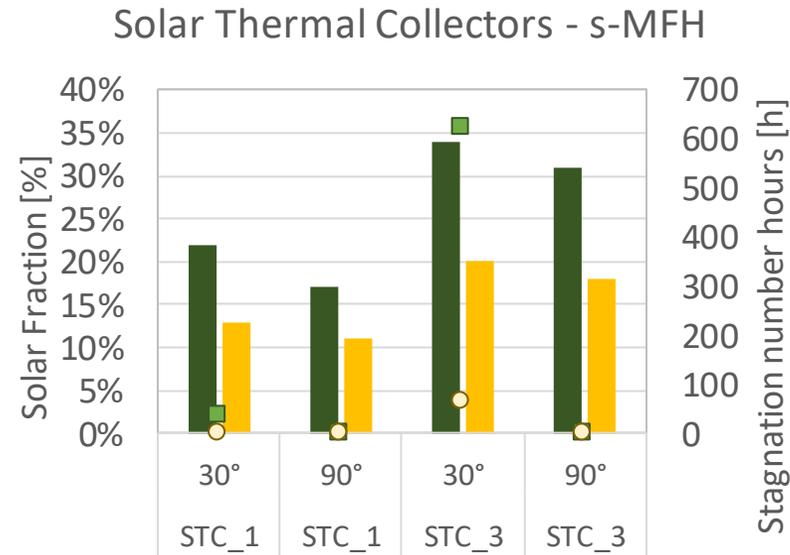


# RESULTS – CASE 1

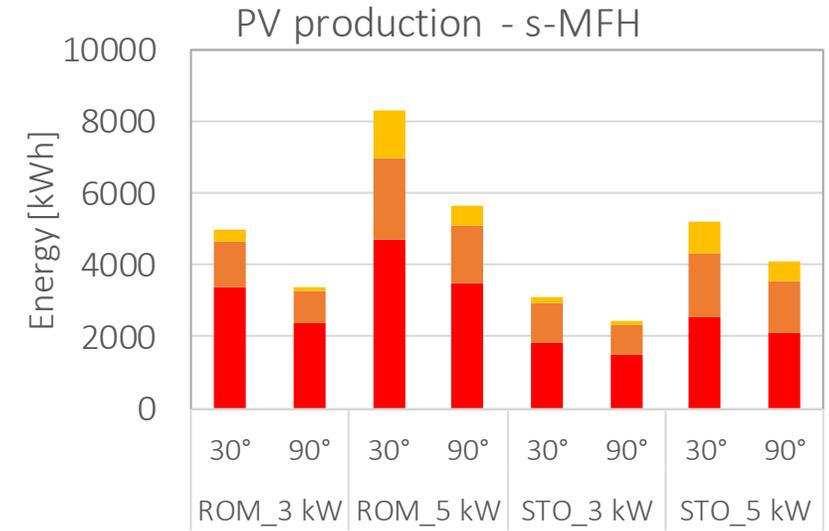
## ST and PV performance with varying field size and tilt angle

Solar Thermal		
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STC_1	m <sup>2</sup>	18.4
STC_2	m <sup>2</sup>	27.6
STC_3	m <sup>2</sup>	36.8

Photovoltaic		
	Unit	s-MFH
PV_1	kWp	3
PV_2	kWp	4
PV_3	kWp	5



■ SF\_ROM ■ SF\_STO ■ Hour\_ROM ○ Hour\_STO



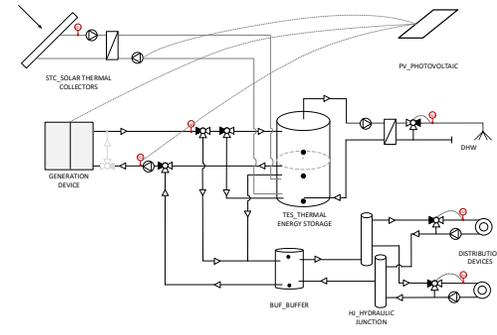
■ PV self HVAC ■ PV self other ■ PV to the grid

ROM – Rome

STO - Stockholm

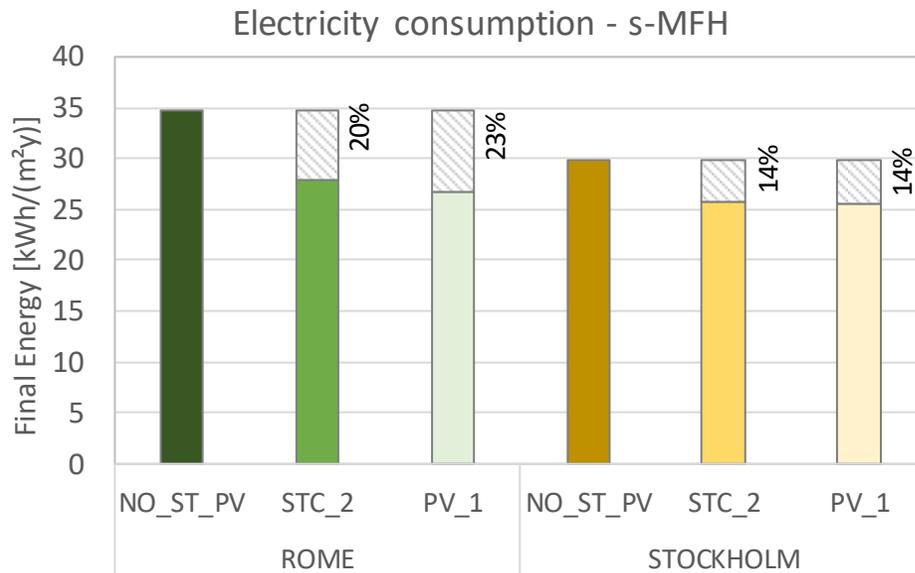
Solar Fraction and stagnation hours referred to the total heating production (**space heating + DHW**)

**PV production and self-consumption** for two different fields size and panel slope



# RESULTS – CASE 1

## ST and PV performance for different sizes and slopes



ROM – Rome

STO – Stockholm

- Slightly higher energy savings in **Southern climates** due to higher cooling loads
- Same energy savings for a solar thermal (STC) or photovoltaic (PV) field in **Northern climates**

*Comparison of similar field areas of STC (27 m<sup>2</sup>) or PV (24 m<sup>2</sup>) in terms of electric energy savings for DHW, heating and cooling uses*

# CASE 2

## Building description

Wooden Residential Building (WRB)

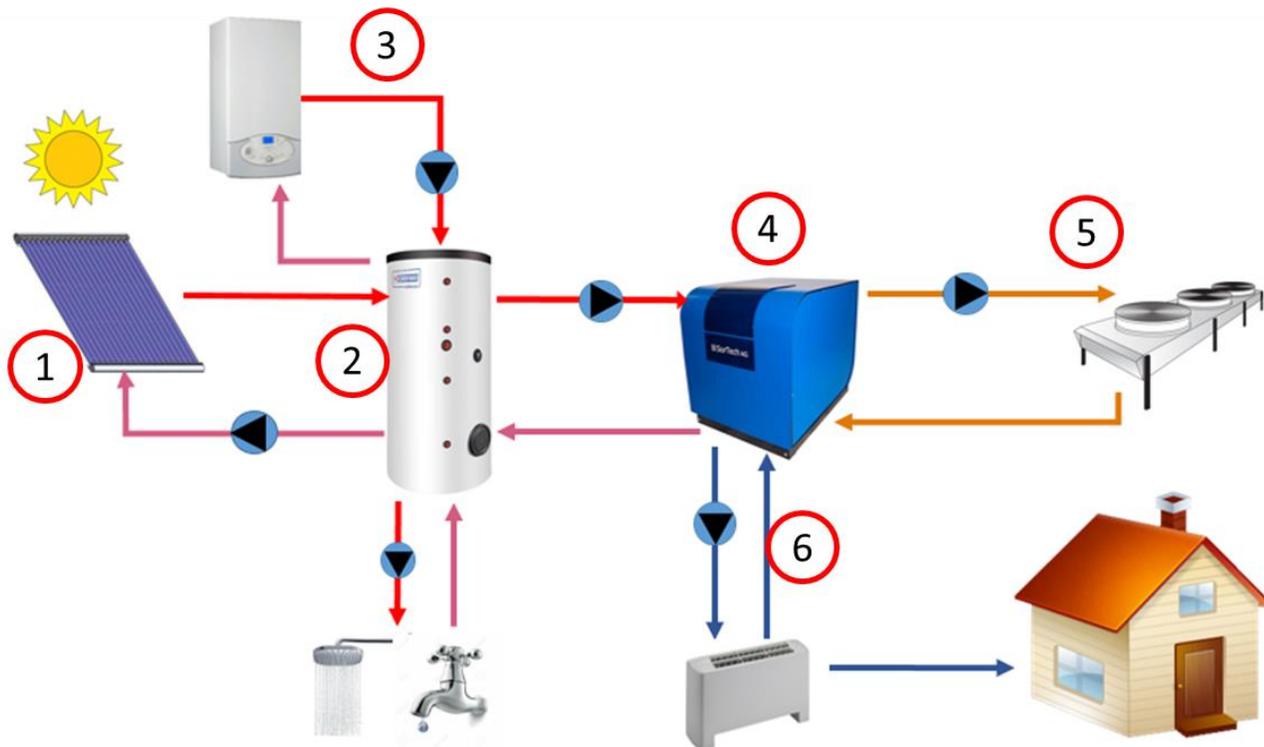


Number of floors	2
Living area per floor	130 m <sup>2</sup>



# CASE 2

## Solar cooling plant layout

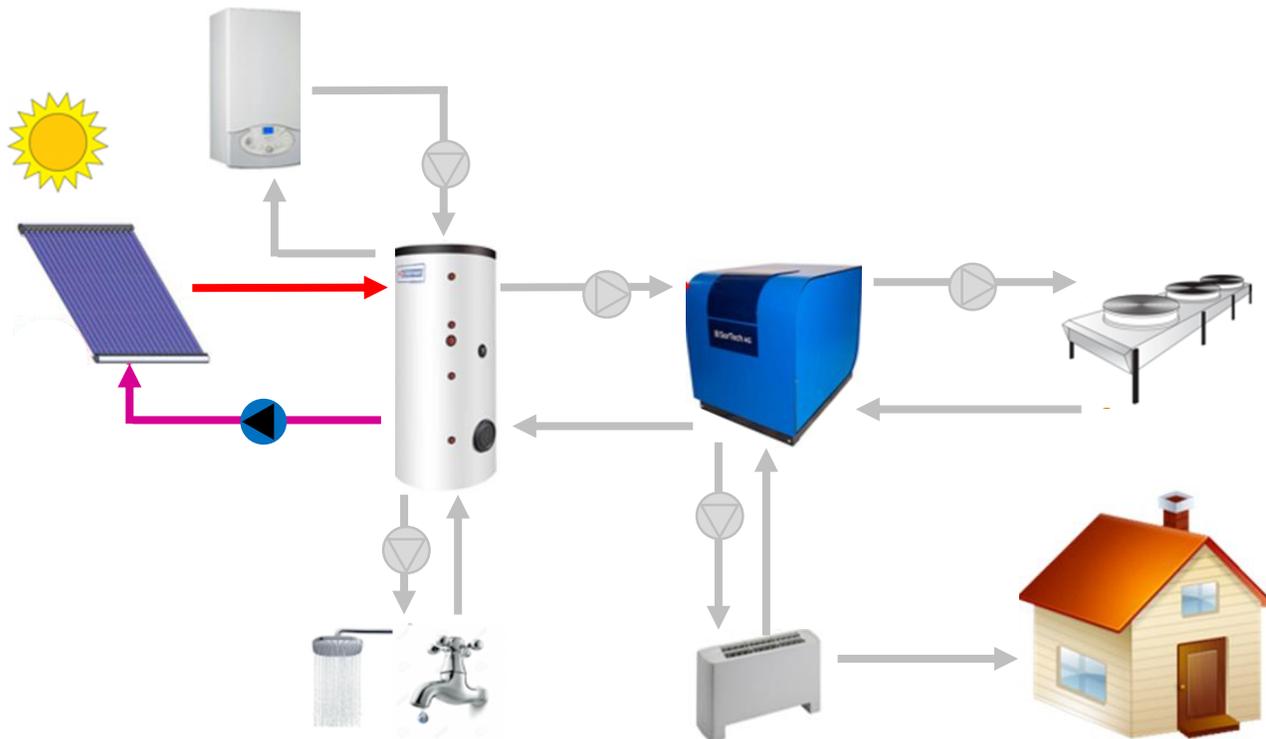


- **Adsorption chiller** for space cooling;
- **Solar collectors (CPC)** for heating and DHW demands
- **Heat rejection** through dry-cooler.

1. Compound Parabolic Collectors (CPC)
2. Storage tank – 1000 l
3. Electric Heater
4. Adsorption chiller – 10 kW
5. Dry cooler
6. Fan coil

# CASE 2

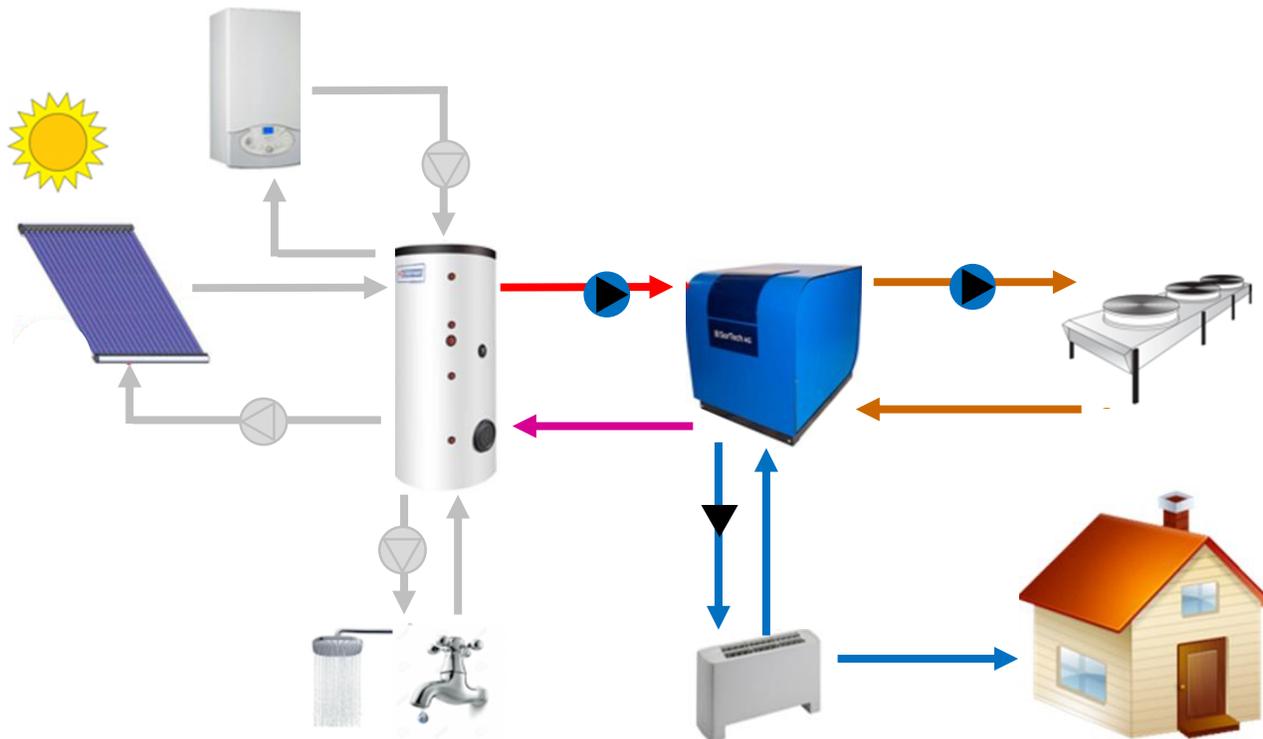
## Working conditions



Running the solar system

# CASE 2

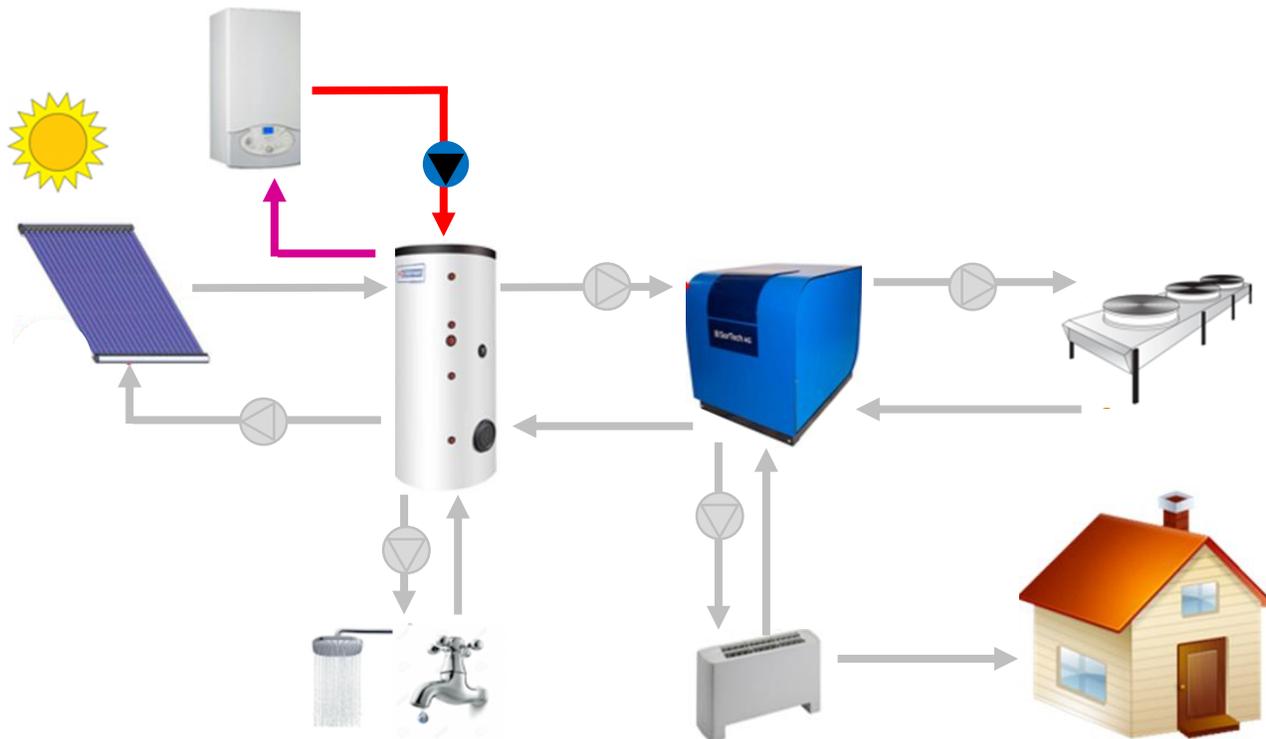
## Working conditions



Space cooling mode

# CASE 2

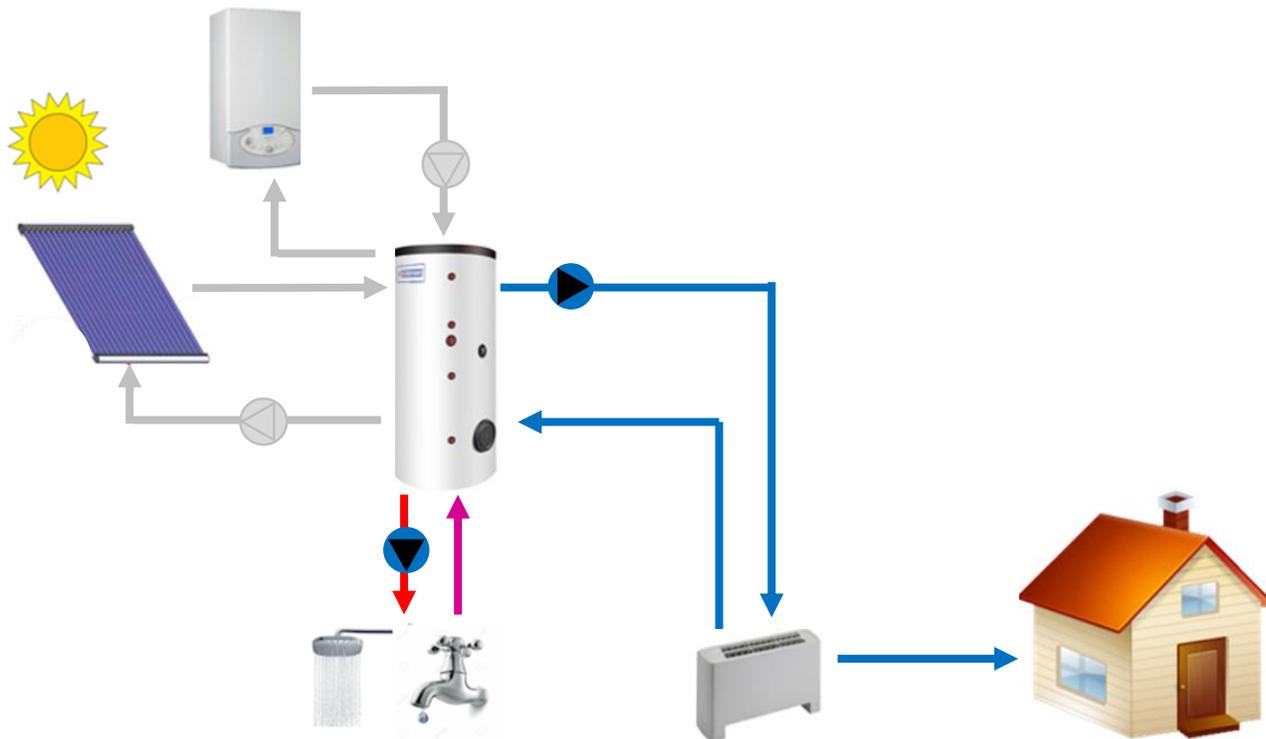
## Working conditions



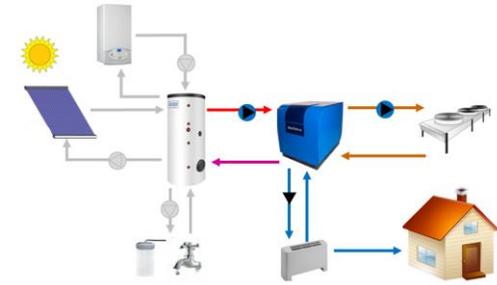
Running the back-up heater

# CASE 2

## Working conditions



Domestic Hot Water and  
space heating

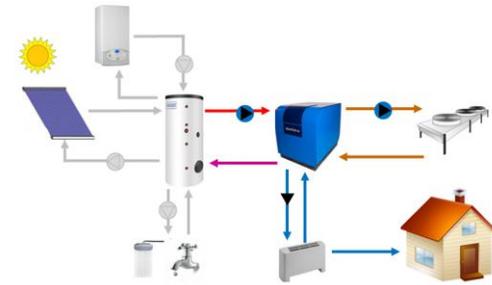


# RESULTS – CASE 2

## Absorption chiller in different climates

	# solar collectors	SPF heating [-]	SPF cooling [-]	SF total [%]	PER total [-]
Freiburg	6	6.7	5.2	51%	1.2
Stuttgart	6	9.5	7.9	56%	1.3
Marseille	6	11.6	9.6	92%	1.9
Messina	8	14.8	12.3	67%	2
Luca	8	14.3	13.0	66%	2
Athens	8	10.4	10.9	69%	2.4
Barcelona	8	12.9	11.7	73%	2.4
Almeria	8	10.7	11.8	66%	1.9
Larnaca	10	11.9	12.7	63%	2.1

- The highest SF is in Marseille where heating and cooling demands are similar;

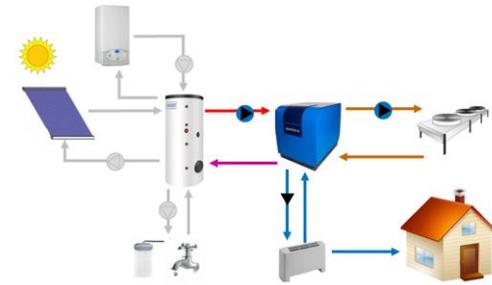


# RESULTS – CASE 2

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Larnaca	10	11.9	12.7	63%	2.1

- The highest SF is in Marseille where heating and cooling demands are similar;
- **Northern climates have low SF** due to small collector size and high heating demand;



# RESULTS – CASE 2

## Absorption chiller in different climates

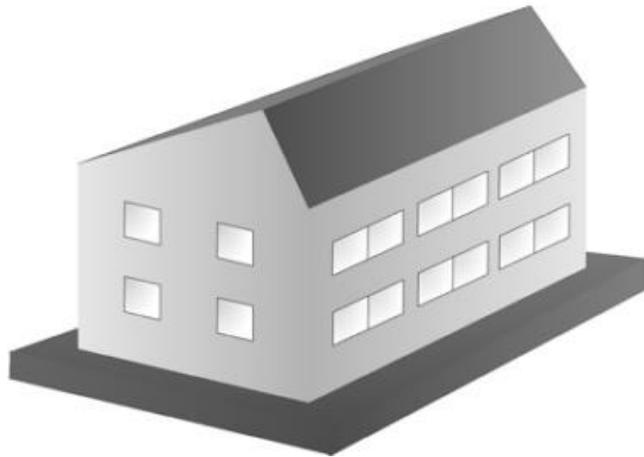
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- The highest SF is in Marseille where heating and cooling demands are similar;
- Northern climates have low SF due to small collector size and high heating demand;
- Although Northern climates are not the best application for adsorption chillers, all the cases have PER (Primary Energy Ratio) > 1 and Solar Fraction > 60%

# CASE 3

## Building description

TheBat Building (Task 44)

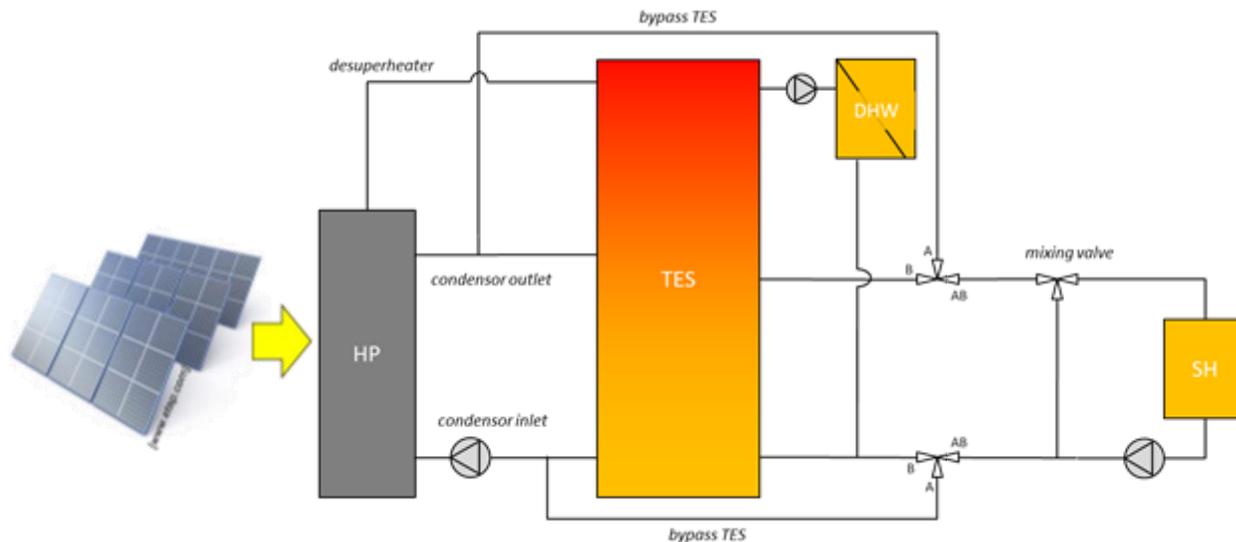


Number of floors	2
Living area per floor	70 m <sup>2</sup>
Yearly heating demand	45 kWh/(m <sup>2</sup> y)

Location: Innsbruck (Austria)

# CASE 3

## Solar cooling plant layout



Use of PV for covering the heat pump consumption:

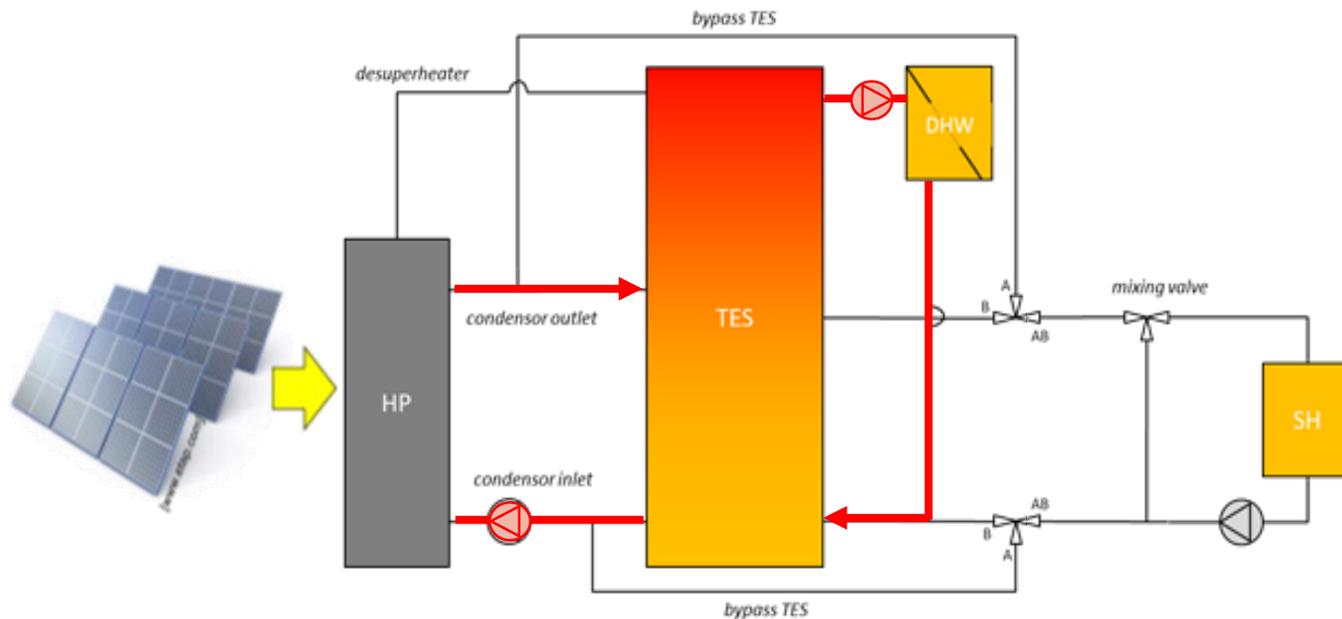
1. SELF consumption;
2. Overheating the TES;
3. Overheating the TABS;
4. Overheating TES and TABS.

1. PV PV panels – 20 m<sup>2</sup> - 40 m<sup>2</sup>
2. HP Heat pump – 10 kW
3. DHW Domestic Hot Water
4. SH Space heating
5. TES Thermal Energy Storage
6. TABS Thermal Activated Building Structure

# CASE 3

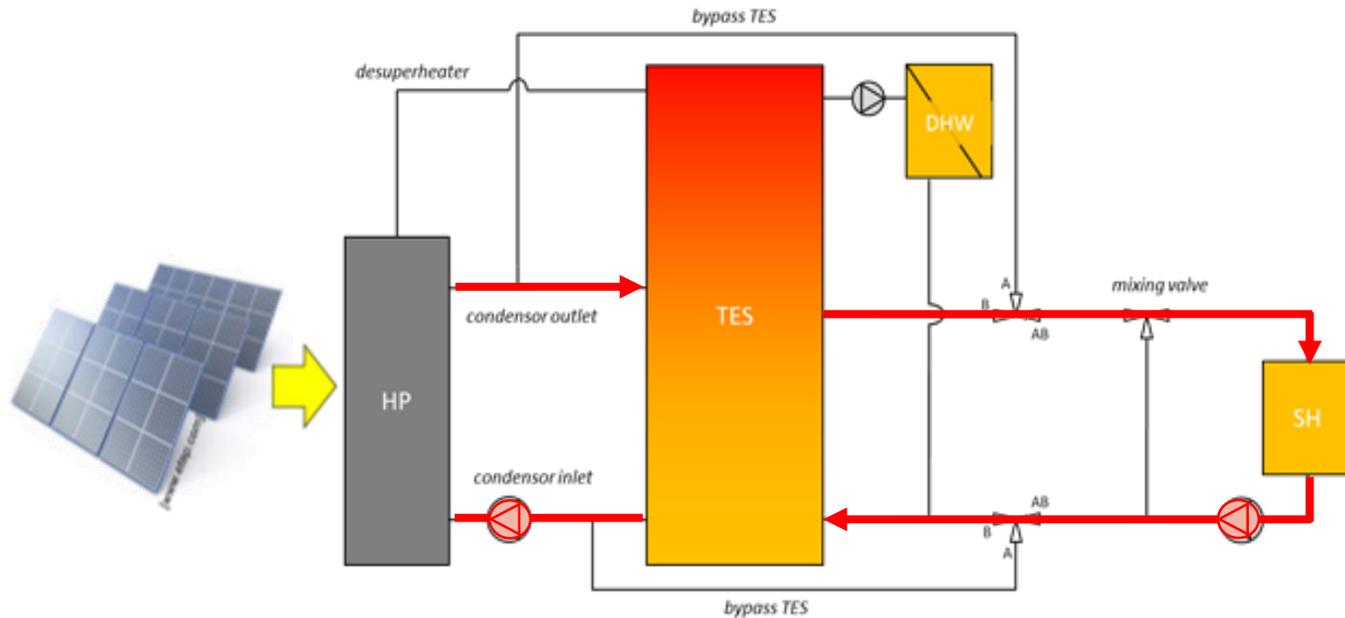
## Working conditions

TES charging for DHW uses



# CASE 3

## Working conditions

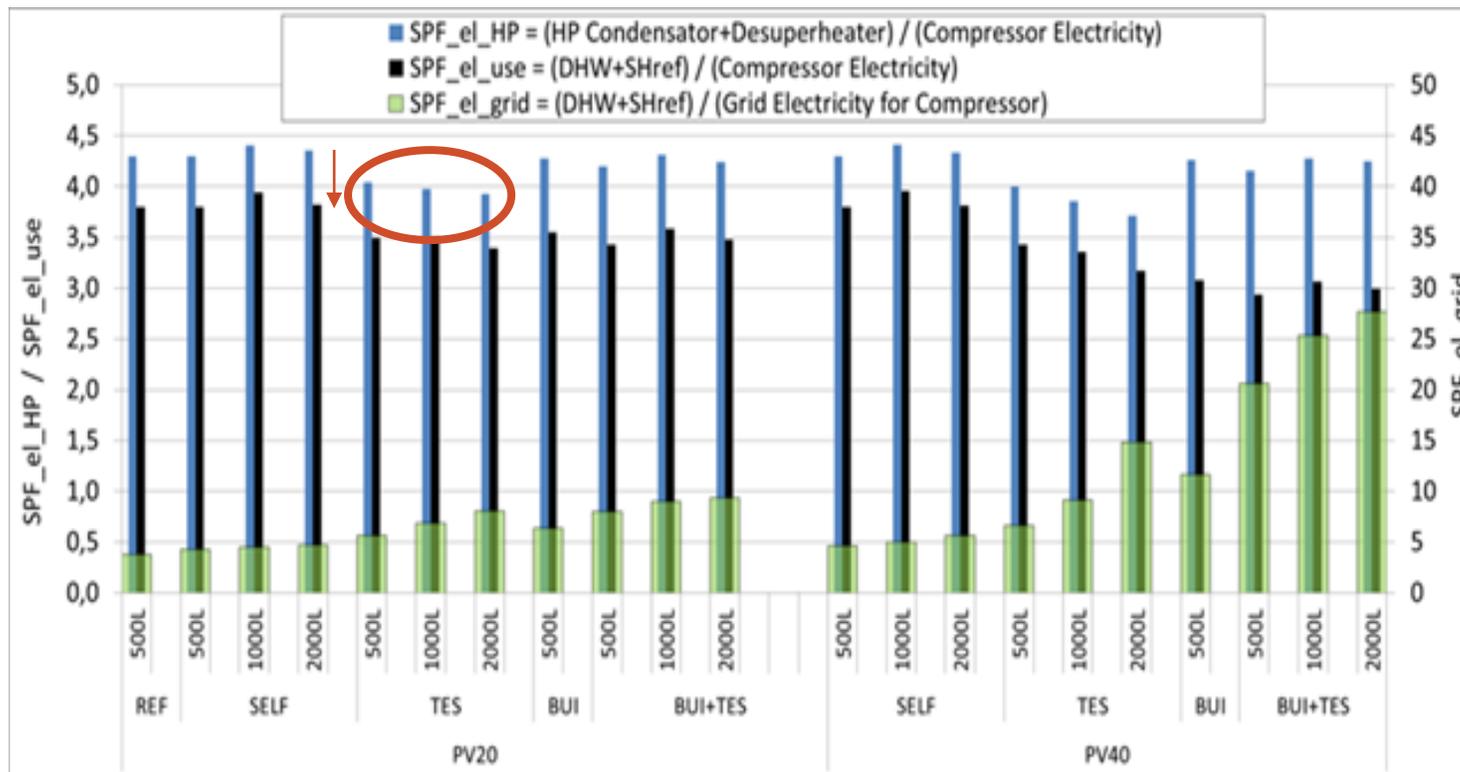
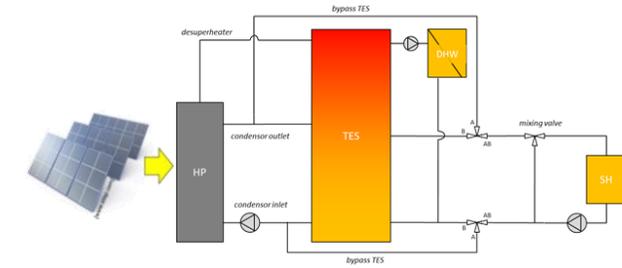


TES charging for space heating use



# RESULTS – CASE 3

## SPF and HP performance at different working conditions

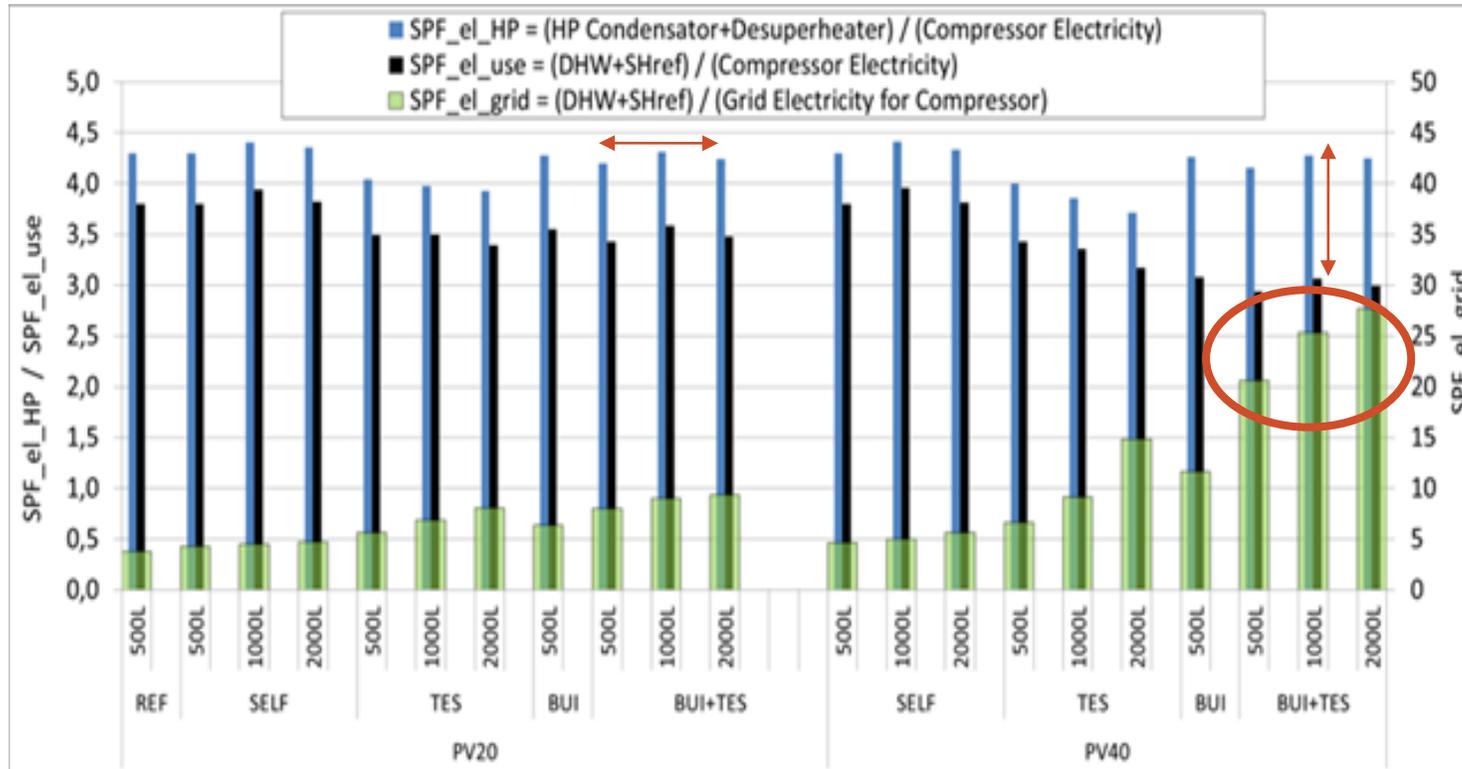
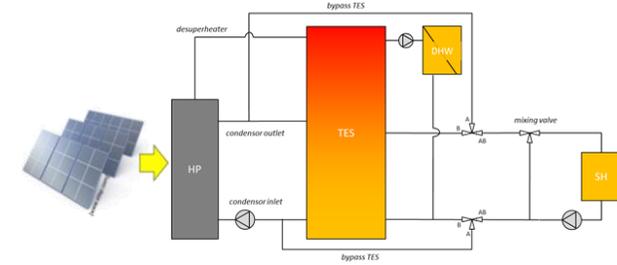


- The strategy of **overheating the TES** reduces the HP performance ( $SPF_{el,HP}$ ) because of the **higher working temperatures**;



# RESULTS – CASE 3

## SPF and HP performance at different working conditions



- The strategy of overheating the TES reduces the HP performance ( $SPF_{el,HP}$ ) because of the higher working temperatures;
- Overheating the BUI and the TES+BUI increases the thermal losses;
- **Bigger PV field area and storage capacity** reduce the used energy from the grid, but **increase energy losses**;
- **Bigger storages** do not significantly improve the system performance.

# CASE 4

## Building description

Multi-family house HVACviaFaçade

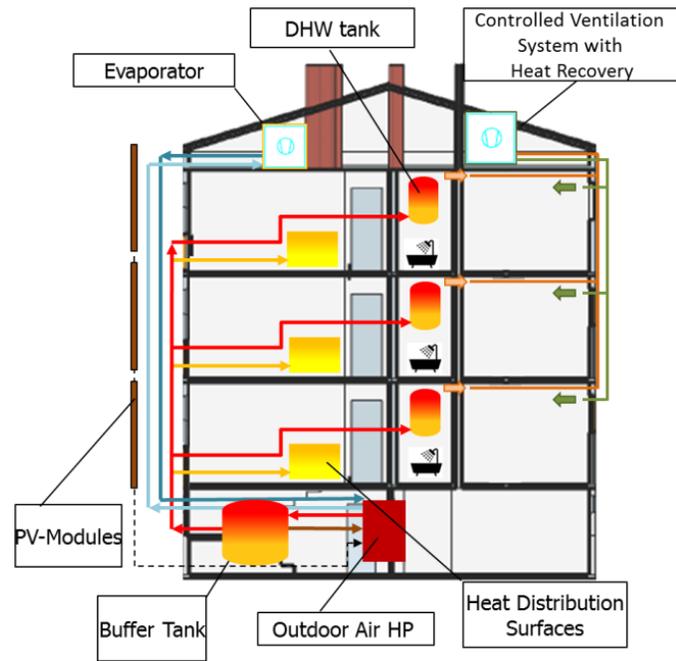


Number of floors	3
Living area per dwelling	50.3 m <sup>2</sup> (average)
Dwellings per floor	4
Yearly heating demand	15 kWh/(m <sup>2</sup> y) – BUI 15
Yearly heating demand	30 kWh/(m <sup>2</sup> y) – BUI 30

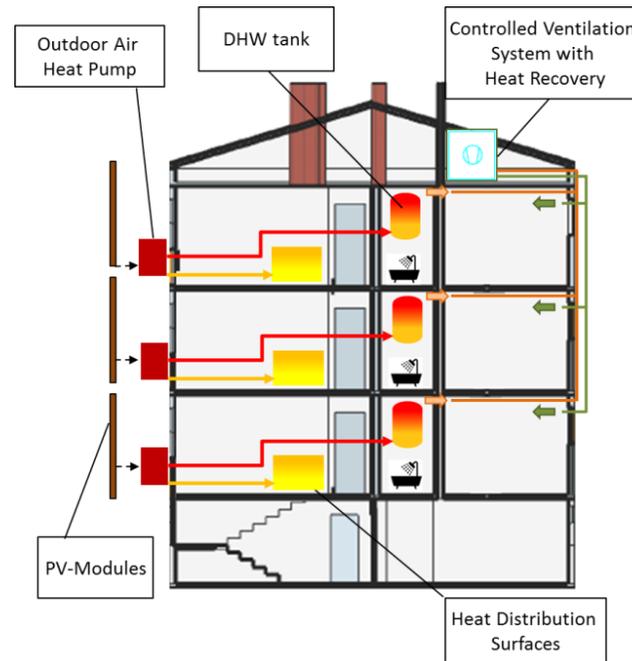
Location: Graz (Austria)

# CASE 4

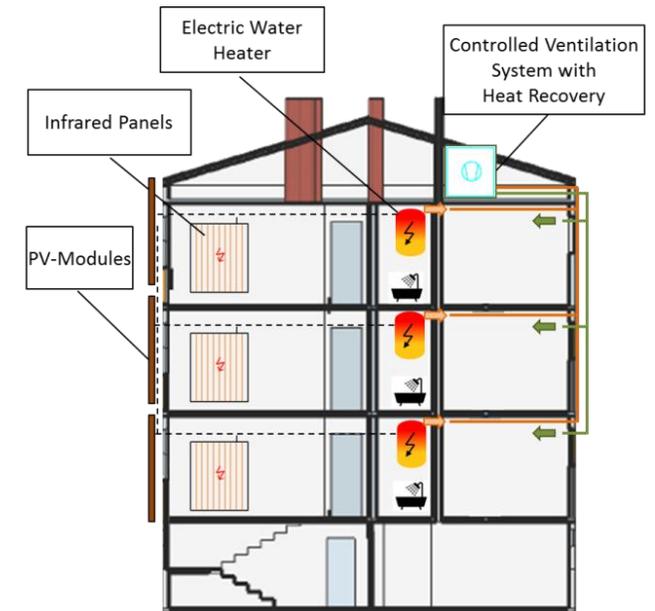
## Layout description



Central outdoor air heat pump



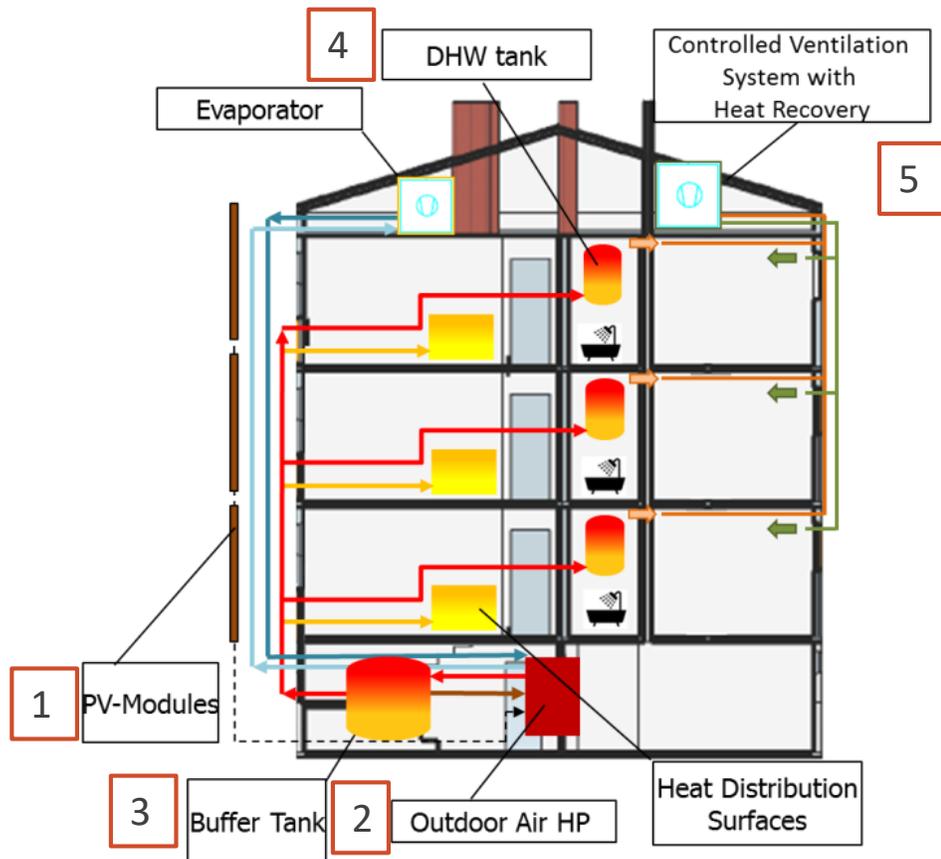
Decentralized outdoor air heat pump



Direct electric heating

# CASE 4 - 1

## Layout description and working conditions



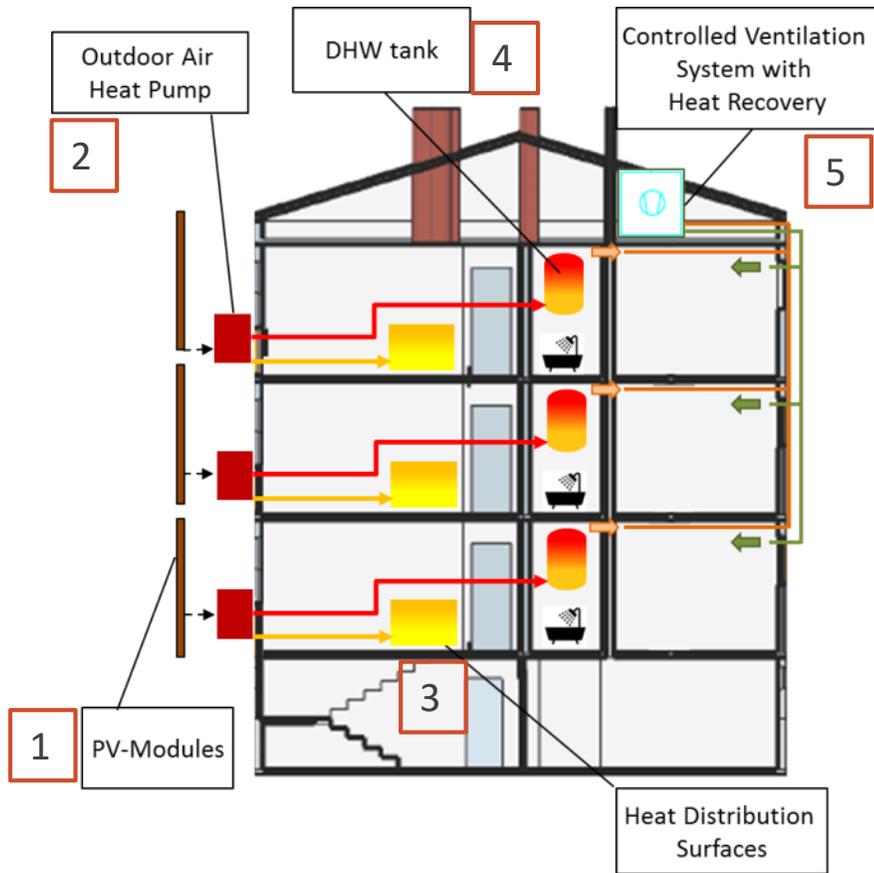
Central outdoor air heat pump

- Maximize PV production for the **centralized heat pump** electric consumption;
- Use of **decentralized storages** for DHW uses;
- Use of **two set temperatures** for the tanks.

1. PV panels 189 m<sup>2</sup> - 15.75 m<sup>2</sup>/dwelling
2. Heat pump 10 kW (BUI 15) – 20 kW (BUI 30)
3. Buffer Tank – 1500 l (BUI 15) – 2000 l (BUI 30)
4. DHW tank – 150 l/dwelling
5. Mechanical Ventilation with Heat Recovery

# CASE 4 - 2

## Layout description and working conditions



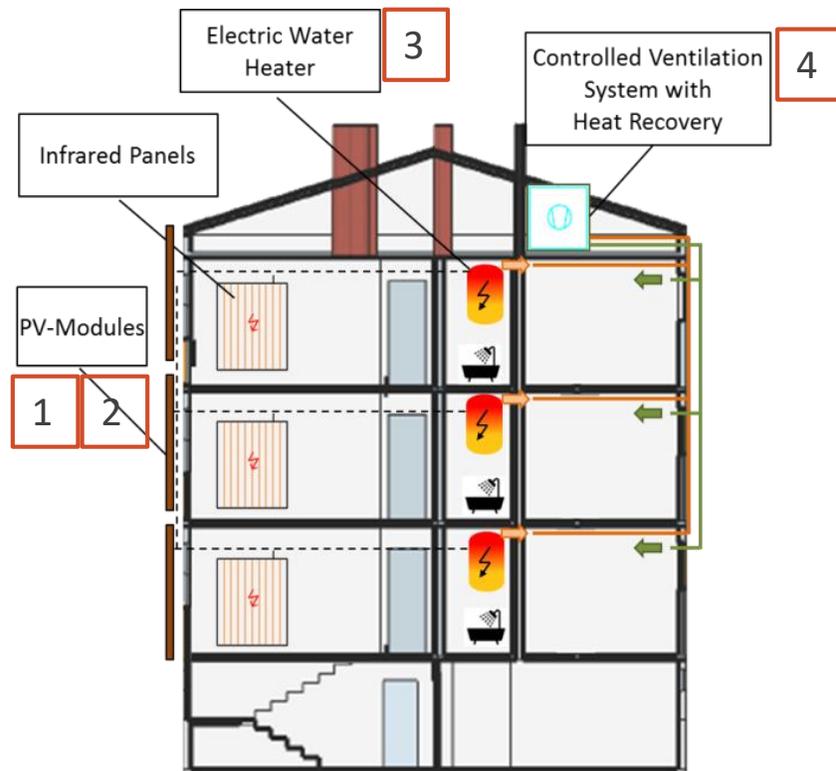
- Maximize PV production for the **decentralized heat pumps** consumption;
- Use of **decentralized storages** for DHW uses;
- Use of **two set temperatures** for the tanks.

1. PV panels 176 m<sup>2</sup> - 14.5 m<sup>2</sup>/dwelling
2. Heat pump 2 kW/dwelling
3. Direct space heating from the heat pump
4. DHW tank – 150 l/dwelling
5. Mechanical Ventilation with Heat Recovery

Decentralized outdoor air heat pump

# CASE 4 - 3

## Layout description and working conditions



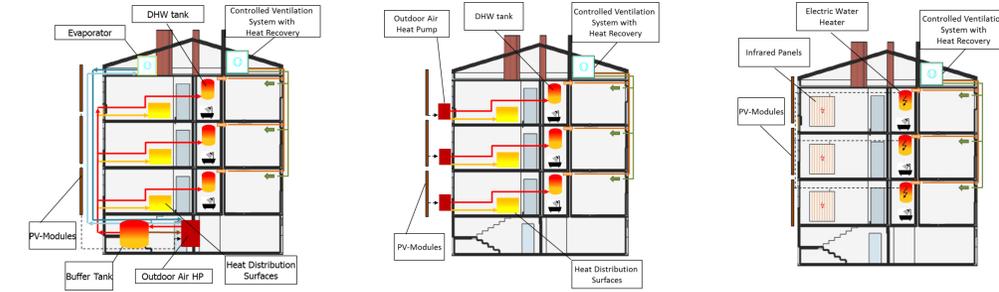
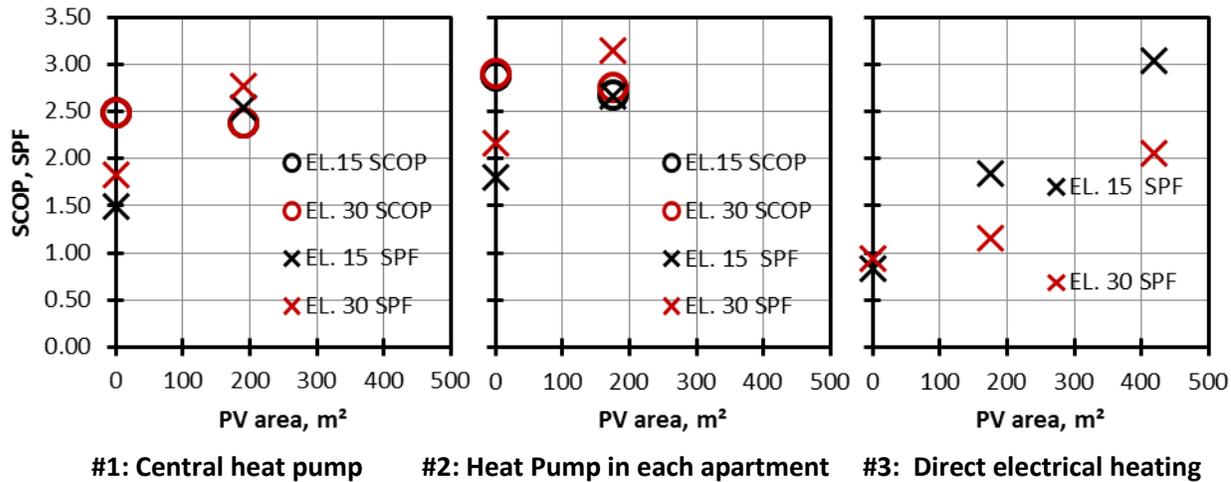
- Maximize **PV production** for self-use;
- Use of **two set temperatures** for the tank;
- Use of the **roof surface** for additional PV panels.

1. PV panels 1 176 m<sup>2</sup> - 14.5 m<sup>2</sup>/dwelling
2. PV panels 2 419 m<sup>2</sup> - 34.9 m<sup>2</sup>/dwelling
3. Electric heater 2.5 kW and 150 l
4. Mechanical Ventilation with Heat Recovery

Direct electric heating

# RESULTS – CASE 4

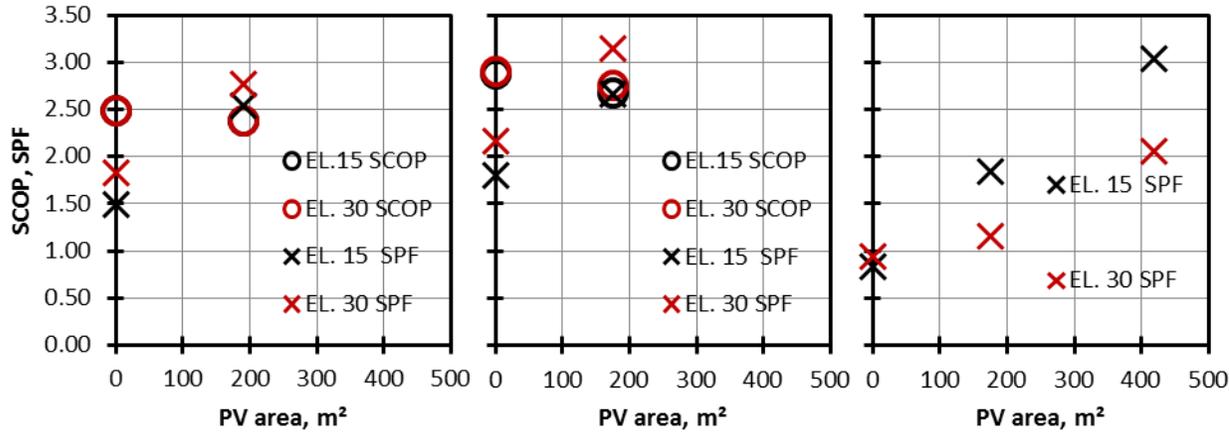
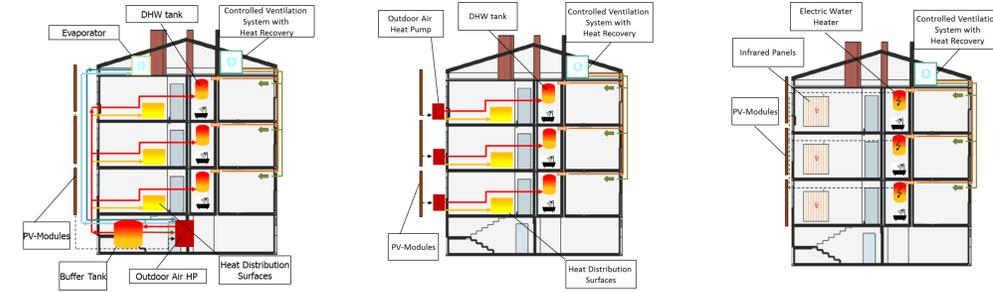
## SPF and SCOP



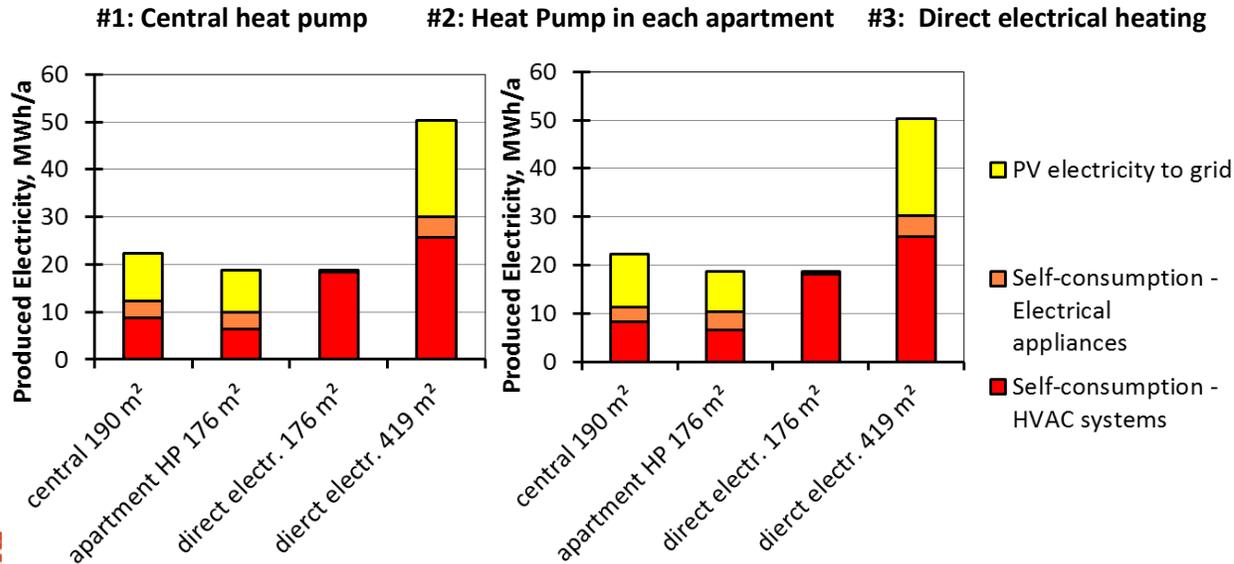
- SCOP is slightly lower in cases with a PV field due to the higher working temperatures
- The highest SPF's are encountered in the decentralized configuration;
- However, a low energy demanding building with a big PV field has a high SPF.

# RESULTS – CASE 4

## SPF and SCOP



- SCOP is slightly lower in cases with a PV field due to the higher working temperatures
- The highest SPF's are encountered in the decentralized configuration;
- However, a low energy demanding building with a big PV field has a high SPF.



- **Self-consumption** accounts for one third to a half of the total production.
- The **excess of electricity fed into the grid** is high in all cases, with exception of the direct heating with small PV field

# CONCLUSIONS

- Solar driven systems can assume different configurations, from the **PV coupled** to a heat pump for heating production to the **integration of solar thermal collectors** for decreasing thermal loads to the use of **sorption chillers** for the cooling loads;
- When designing a solar energy system, the **solar field size is key**, in fact bigger solar thermal fields can cause **stagnation problems** and in PV systems the **self-consumption** can be only a **small fraction of the produced energy** (20% to 30%);
- Solar technologies have **good results in terms of solar fraction and SPF also in northern climates** thanks to the longer winter season and the inclination of solar radiation in this period.
- The use of **thermal storages** can help to **maximize the use of solar energy** also in combination with PV systems.

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**THANK YOU**

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